

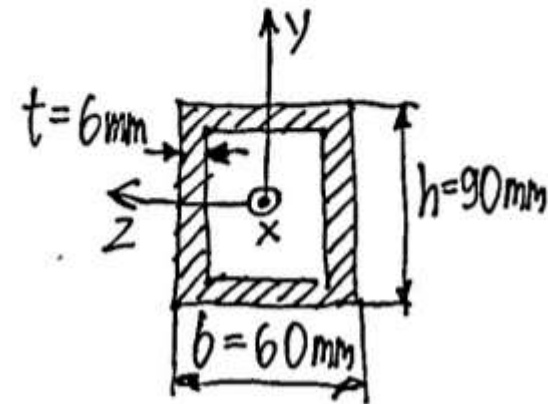
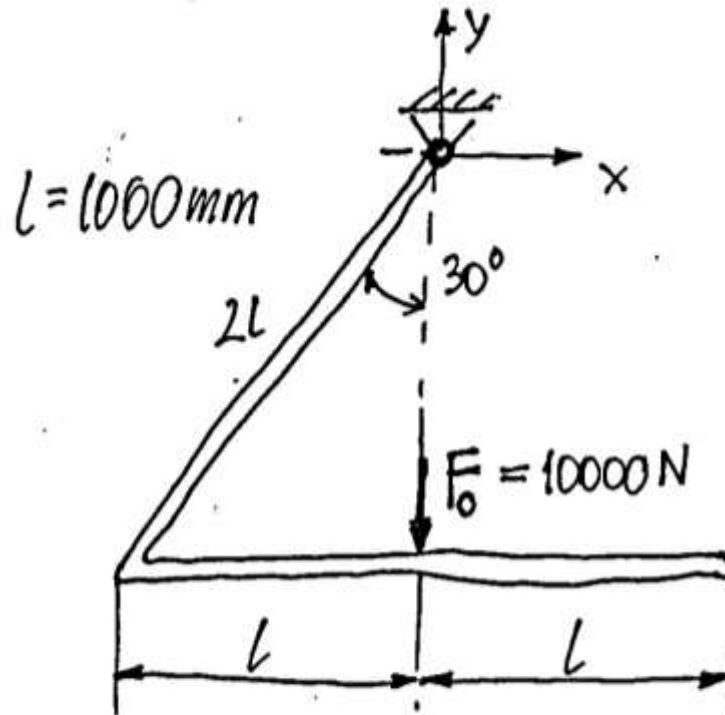


Finite element method (FEM1)

Lecture 10C. Frames - examples

05.2025

Example: Build a 2D FEM model of the frame. Determine nodal displacements, stresses, internal forces and reactions. Check equilibrium conditions.



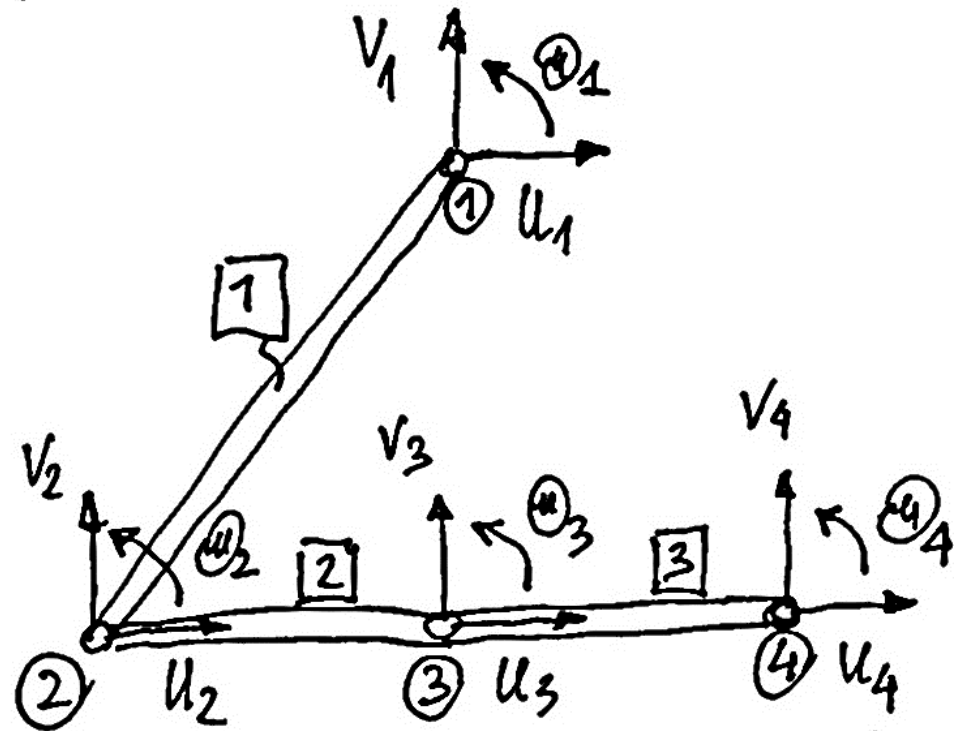
$$A = bh - (b-2t)(h-2t) = 1656 \text{ mm}^2$$

$$J_z = \frac{bh^3}{12} - \frac{(b-2t)(h-2t)^3}{12} = 1.7468 \cdot 10^6 \text{ mm}^4$$

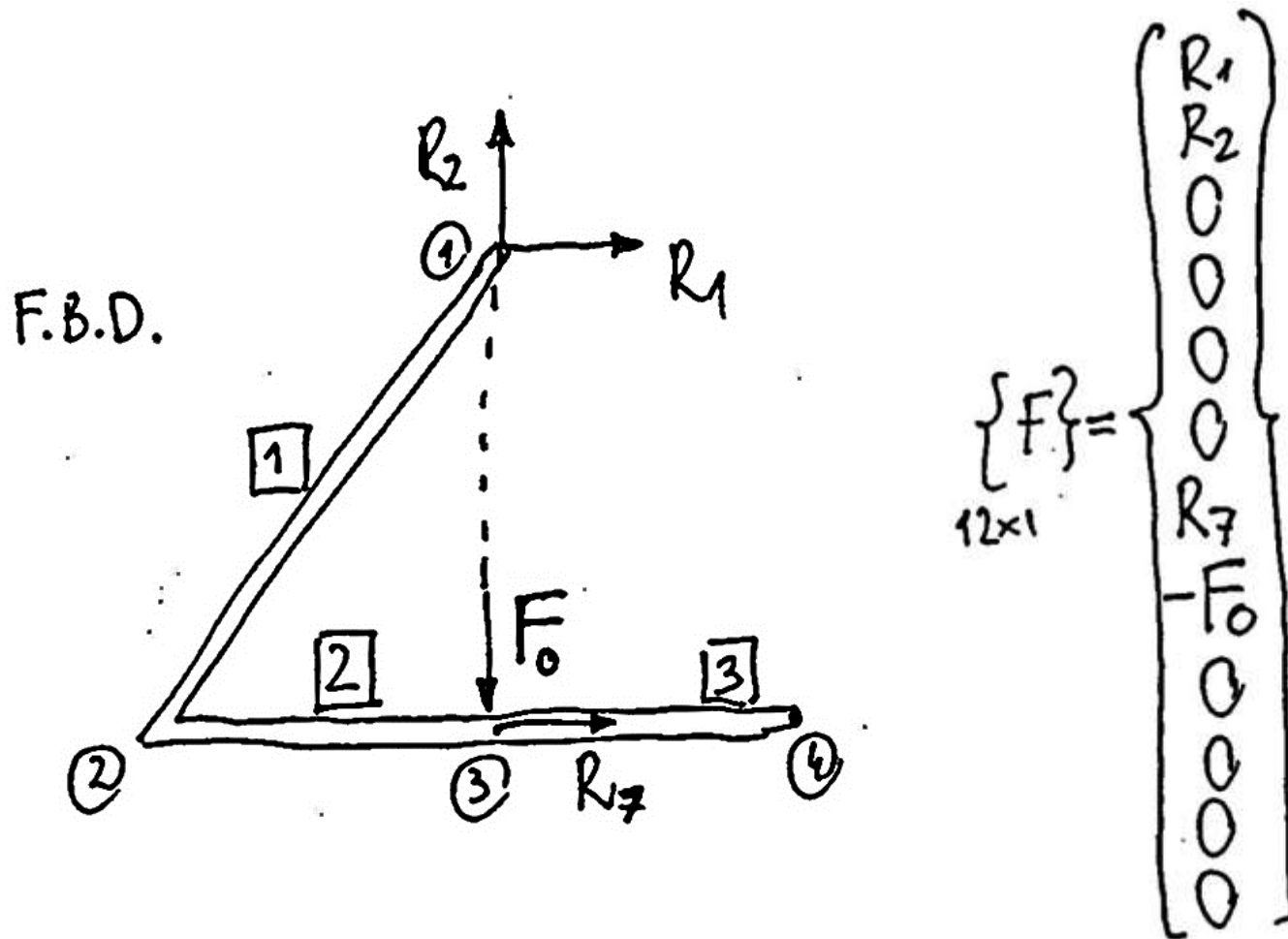
Nodal parameters:

$$\{q\}_{12 \times 1} = \begin{Bmatrix} u_1 \\ v_1 \\ \textcircled{1}_1 \\ u_2 \\ v_2 \\ \textcircled{2}_2 \\ u_3 \\ v_3 \\ \textcircled{3}_3 \\ u_4 \\ v_4 \\ \textcircled{4}_4 \end{Bmatrix}$$

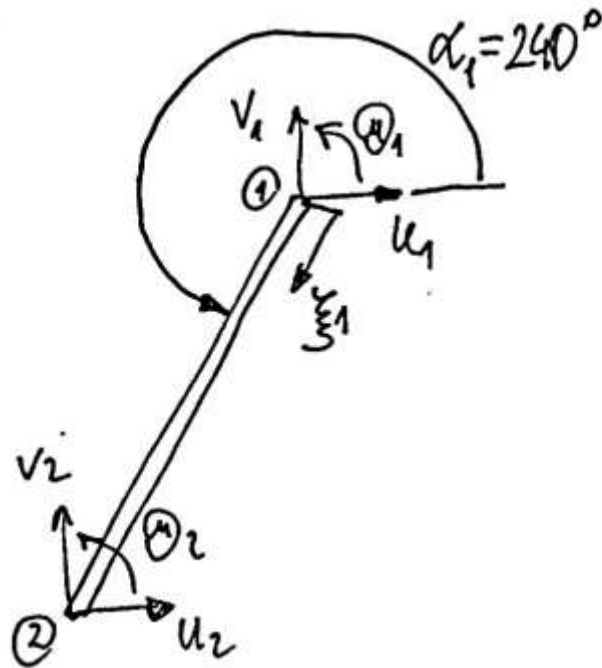
$$\begin{aligned} u_1 &= 0 \\ v_1 &= 0 \\ u_3 &= 0 \end{aligned}$$



Loads and reactions:



1



Stiffness matrices of element 1:

$$C_1 = -\frac{1}{2}$$

$$S_1 = -\frac{\sqrt{3}}{2}$$

$$[q]_1 = [u_1, v_1, \theta_1, u_2, v_2, \theta_2]$$

1x6

$$[k]_1 =$$

6x6

$$\begin{bmatrix} \frac{EA}{2L} & 0 & 0 & -\frac{EA}{2L} & 0 & 0 \\ 0 & \frac{12EJ_2}{8L^3} & \frac{6EJ_2}{4L^2} & 0 & -\frac{12EJ_2}{8L^3} & \frac{6EJ_2}{4L^2} \\ 0 & \frac{6EJ_2}{4L^2} & \frac{4EJ_2}{2L} & 0 & -\frac{6EJ_2}{4L^2} & \frac{2EJ_2}{2L} \\ -\frac{EA}{2L} & 0 & 0 & \frac{EA}{2L} & 0 & 0 \\ 0 & -\frac{12EJ_2}{8L^3} & -\frac{6EJ_2}{4L^2} & 0 & \frac{12EJ_2}{8L^3} & -\frac{6EJ_2}{4L^2} \\ 0 & \frac{6EJ_2}{4L^2} & \frac{2EJ_2}{2L} & 0 & -\frac{6EJ_2}{4L^2} & \frac{4EJ_2}{2L} \end{bmatrix}$$

Transformation matrix of element 1:

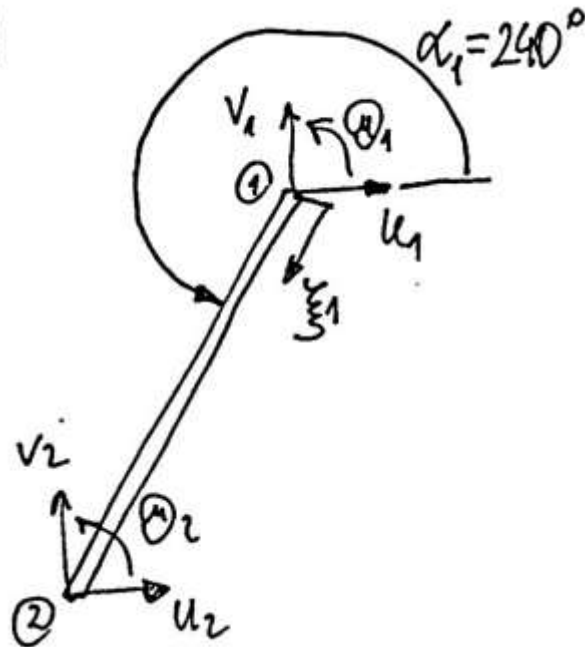
$$[T_f]_e = \begin{bmatrix} c & s & 0 & 0 & 0 & 0 \\ -s & c & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & c & s & 0 \\ 0 & 0 & 0 & -s & c & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \end{bmatrix}$$

$$[T_f]_1 = \begin{bmatrix} -\frac{1}{2} & -\frac{\sqrt{3}}{2} & 0 & 0 & 0 & 0 \\ \frac{\sqrt{3}}{2} & -\frac{1}{2} & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & -\frac{1}{2} & -\frac{\sqrt{3}}{2} & 0 \\ 0 & 0 & 0 & \frac{\sqrt{3}}{2} & -\frac{1}{2} & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \end{bmatrix}$$

$$[T_f]_1^T = \begin{bmatrix} -\frac{1}{2} & \frac{\sqrt{3}}{2} & 0 & 0 & 0 & 0 \\ -\frac{\sqrt{3}}{2} & -\frac{1}{2} & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & -\frac{1}{2} & \frac{\sqrt{3}}{2} & 0 \\ 0 & 0 & 0 & \frac{\sqrt{3}}{2} & -\frac{1}{2} & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \end{bmatrix}$$

Extended transformation matrix of element 1:

1



$$[k_g]_1 = [T_f]_1^T [k]_1 [T_f]_1$$

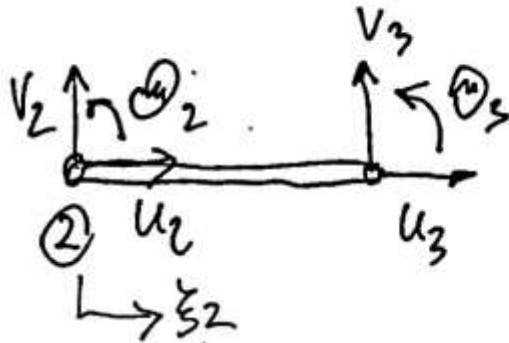
6×6 6×6 6×6 6×6

$$[k_g]_1^* = \begin{bmatrix} [k_g]_1 & [0] \\ [0] & [0] \end{bmatrix}$$

12×12 6×6 6×6 6×6

Stiffness matrices of element 2:

[2]



$$\alpha_2 = 0^\circ, \quad c_2 = 1, \quad s_2 = 0$$

$$[q]_2 = [u_2, v_2, \theta_2, u_3, v_3, \theta_3]$$

1×6

$$[K]_2 = \begin{bmatrix} \frac{EA}{L} & 0 & 0 & -\frac{EA}{L} & 0 & 0 \\ 0 & \frac{12EJ_2}{L^3} & \frac{6EJ_2}{L^2} & 0 & -\frac{12EJ_2}{L^3} & \frac{6EJ_2}{L^2} \\ 0 & \frac{6EJ_2}{L^2} & \frac{4EJ_2}{L} & 0 & -\frac{6EJ_2}{L^2} & \frac{2EJ_2}{L} \\ -\frac{EA}{L} & 0 & 0 & \frac{EA}{L} & 0 & 0 \\ 0 & -\frac{12EJ_2}{L^3} & -\frac{6EJ_2}{L^2} & 0 & \frac{12EJ_2}{L^3} & -\frac{6EJ_2}{L^2} \\ 0 & \frac{6EJ_2}{L^2} & \frac{2EJ_2}{L} & 0 & -\frac{6EJ_2}{L^2} & \frac{4EJ_2}{L} \end{bmatrix}$$

6×6

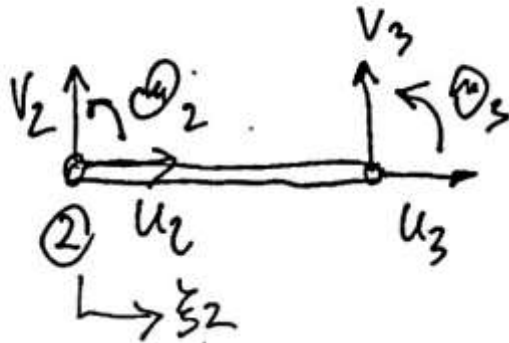
Transformation matrix of element 2:

$$[\bar{T}_f]_2 = \begin{bmatrix} 1 & 0 & 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \end{bmatrix} = [\bar{T}_f]_2^T$$

$$[K_g]_2 = [\bar{T}_f]_2^T [K]_2 [\bar{T}_f]_2 = [k]_2$$

Extended transformation matrix of element 2:

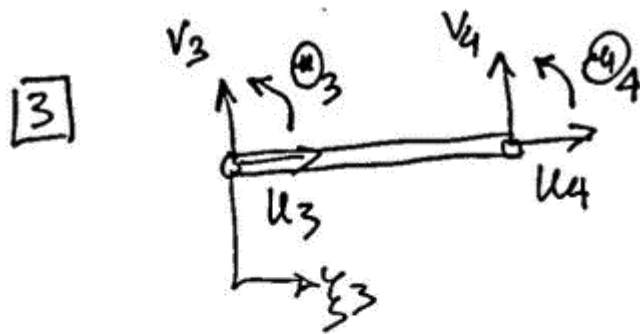
2



$$[K_g]_2^* =$$

$$= \begin{bmatrix} [0]_{3 \times 3} & [0]_{3 \times 6} & [0]_{3 \times 3} \\ [0]_{6 \times 3} & [K_g]_2 & [0]_{6 \times 3} \\ [0]_{3 \times 3} & [0]_{3 \times 6} & [0]_{3 \times 3} \end{bmatrix}$$

Extended transformation matrix of element 3:



$$\alpha_3 = 0^\circ, \quad C_3 = 1, \quad S_3 = 0$$

$$[q_g]_3 = [u_3, v_3, \theta_3, u_4, v_4, \theta_4]$$

1×6

$$[K]_3 = [k]_2, \quad [T_f]_3 = [T_f]_3^T = [\bar{T}_f]_2$$

$6 \times 6 \quad 6 \times 6 \quad 6 \times 6$

$$[k_g]_3 = [k_g]_2 = [k]_3$$

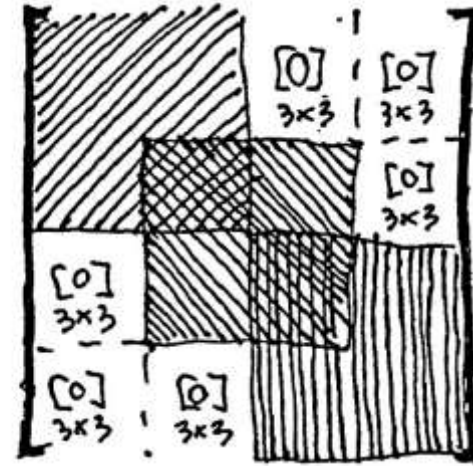
$6 \times 6 \quad 6 \times 6 \quad 6 \times 6$

$$[k_g]_3^* =$$

$$\begin{bmatrix} [0]_{6 \times 6} & [0]_{6 \times 6} \\ [0]_{6 \times 6} & [k_g]_3^* \end{bmatrix}$$

Global Stiffness Matrix:

$$[K]_{12 \times 12} = [K_g]_1^* + [K_g]_2^* + [K_g]_3^* =$$



System of equations:

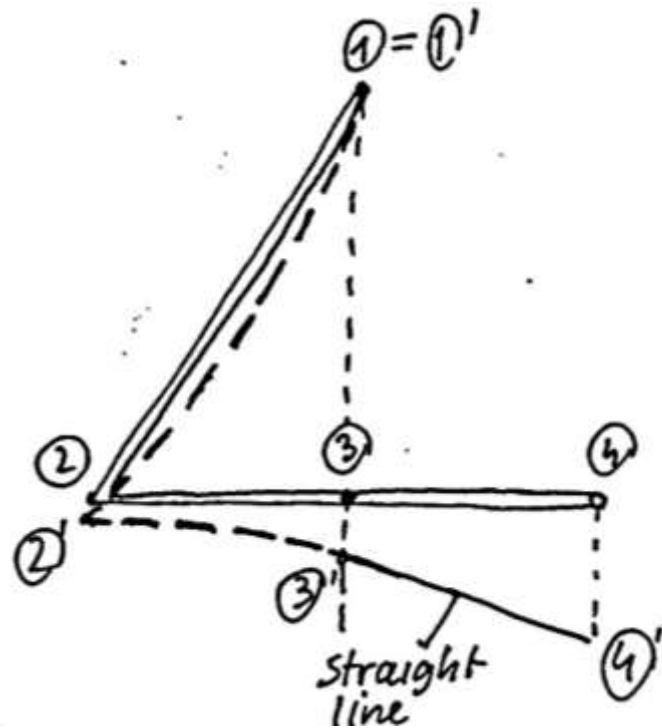
$$[K]_{12 \times 12} \cdot \{q\}_{12 \times 1} = \{F\}_{12 \times 1} + \text{boundary conditions}$$

$$u_1 = 0, v_1 = 0, u_3 = 0$$

$$[K]_{9 \times 9} \cdot \begin{Bmatrix} u_1 \\ u_2 \\ v_2 \\ u_2 \\ v_3 \\ u_3 \\ u_4 \\ v_4 \\ u_4 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \\ 0 \\ 0 \\ -10000 \\ 0 \\ 0 \\ 0 \\ 0 \end{Bmatrix}$$

$$([K]_{9 \times 9} = [K]_{12 \times 12} \text{ without rows and columns: no. 1, 2, 7})$$

Nodal displacements:



$$\begin{Bmatrix} q_1 \\ q_2 \\ q_3 \\ q_4 \\ q_5 \\ q_6 \\ q_7 \\ q_8 \\ q_9 \end{Bmatrix}_{9 \times 1} = \begin{bmatrix} K \end{bmatrix}_{9 \times 9}^{-1} \cdot \begin{Bmatrix} F_1 \\ F_2 \\ F_3 \\ F_4 \\ F_5 \\ F_6 \\ F_7 \\ F_8 \\ F_9 \end{Bmatrix}_{9 \times 1} = \begin{Bmatrix} 0.00956 & 0 & -0.0604 & -0.01907 & -28.6692 & -0.0334 & 0 & -62.0486 & -0.0334 \end{Bmatrix} \begin{matrix} (\text{rad}) \\ (\text{mm}) \\ (\text{mm}) \\ (\text{rad}) \\ (\text{mm}) \\ (\text{rad}) \\ (\text{mm}) \\ (\text{mm}) \\ (\text{rad}) \end{matrix} = \begin{Bmatrix} \textcircled{4}_1 \\ u_2 \\ v_2 \\ \textcircled{4}_2 \\ v_3 \\ \textcircled{4}_3 \\ u_4 \\ v_4 \\ \textcircled{4}_4 \end{Bmatrix}$$

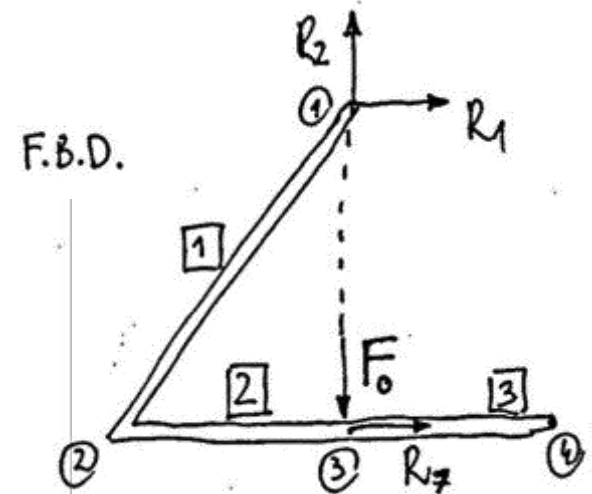
Reactions:

$$\begin{array}{c}
 [K] \cdot \{q\} = \begin{Bmatrix} R_1 \\ R_2 \\ 0 \\ 0 \\ 0 \\ 0 \\ R_7 \\ -F_0 \\ 0 \\ 0 \\ 0 \\ 0 \end{Bmatrix} \\
 \begin{matrix} 12 \times 12 & 12 \times 1 \end{matrix}
 \end{array}
 \Rightarrow R_1 = 0 \text{ N}$$

$$\Rightarrow R_2 = 10000 \text{ N}$$

$$\Rightarrow R_7 = 0 \text{ N}$$

(additional
constraint $u_3 = 0$
did not change the
deformation)



Displacements of element 1 in the local coordinate system:

[1]

$$\begin{Bmatrix} q_1 \\ q_2 \\ q_3 \\ q_4 \\ q_5 \\ q_6 \end{Bmatrix}_1 = \underset{6 \times 6}{[T_f]}_1 \cdot \underset{6 \times 1}{\{q_g\}}_1 = \underset{6 \times 6}{[T_f]}_1 \cdot \begin{Bmatrix} u_1 \\ v_1 \\ \theta_1 \\ u_2 \\ v_2 \\ \theta_2 \end{Bmatrix}_1 =$$

$$= \begin{bmatrix} -\frac{1}{2} & -\frac{\sqrt{3}}{2} & 0 & 0 & 0 & 0 \\ \frac{\sqrt{3}}{2} & -\frac{1}{2} & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & -\frac{1}{2} & -\frac{\sqrt{3}}{2} & 0 \\ 0 & 0 & 0 & \frac{\sqrt{3}}{2} & -\frac{1}{2} & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \end{bmatrix} \cdot \begin{Bmatrix} 0 \\ 0 \\ 0.00956 \\ 0 \\ -0.0604 \\ -0.01907 \end{Bmatrix}_1 = \begin{Bmatrix} 0 \\ 0 \\ 0.00956 \\ 0.0523 \\ 0.0302 \\ -0.01907 \end{Bmatrix}_1$$

Internal stresses and forces in element 1:

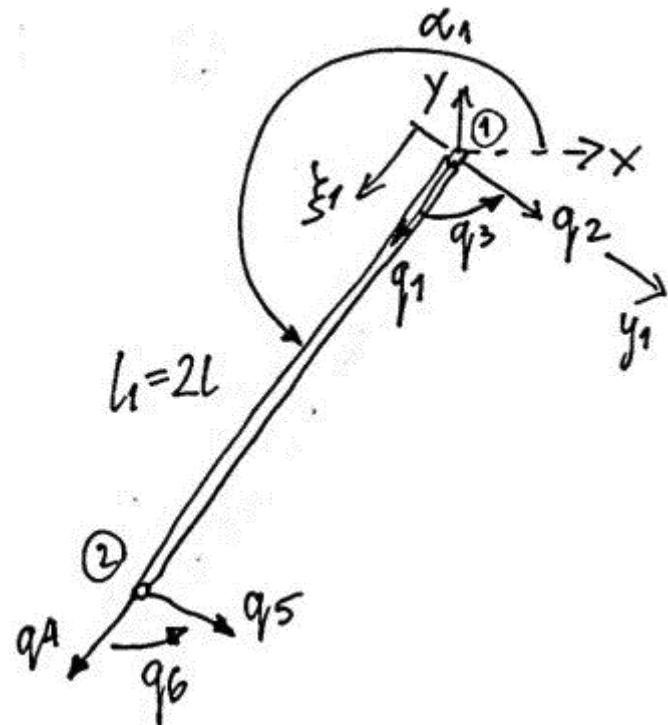
1

$$\begin{Bmatrix} q_1 \\ q_2 \\ q_3 \\ q_4 \\ q_5 \\ q_6 \end{Bmatrix}_1 = \begin{Bmatrix} 0 \\ 0 \\ 0.00956 \\ 0.0523 \\ 0.0302 \\ -0.01907 \end{Bmatrix}_1$$

Tension in element 1:

$$\sigma_1 = \frac{E}{2L} (q_4 - q_1) = 5.23 \text{ MPa}$$

$$N_1 = \sigma_1 \cdot A = 8660.254 \text{ N}$$



Internal forces in element 1 due to bending:

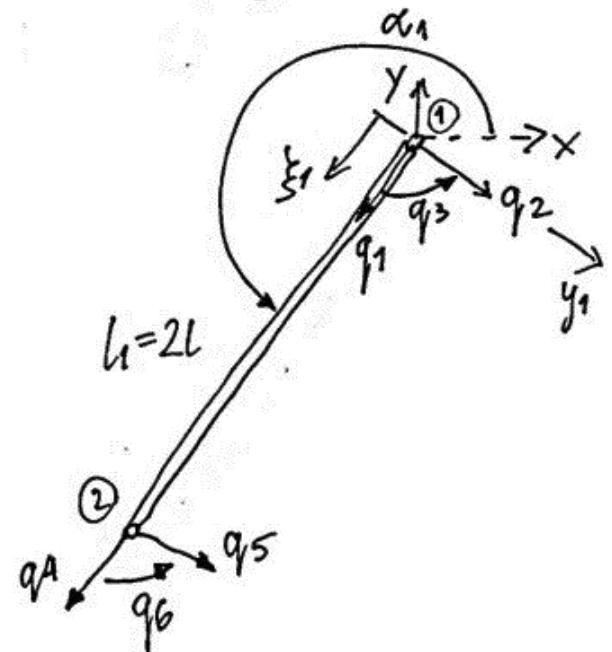
$$M_{z_1}(\xi_1) = EJ_2 \cdot W_1'' = EJ_2 \cdot \underset{1 \times 4}{[N'']} \cdot \begin{Bmatrix} q_2 \\ q_3 \\ q_5 \\ q_6 \end{Bmatrix}_1 =$$

$$= EJ_2 \left[-\frac{6}{l_1^2} + \frac{12}{l_1^3} \xi_1, -\frac{4}{l_1} + \frac{6}{l_1^2} \xi_1, \frac{6}{l_1^2} - \frac{12}{l_1^3} \xi_1, -\frac{2}{l_1} + \frac{6}{l_1^2} \xi_1 \right] \cdot \begin{Bmatrix} q_1 \\ q_2 \\ q_5 \\ q_6 \end{Bmatrix}_1 \Rightarrow$$

$$\Rightarrow M_{z_1}(0) = 0, \quad M_{z_1}(2l) = -1 \cdot 10^7 \text{ Nmm}$$

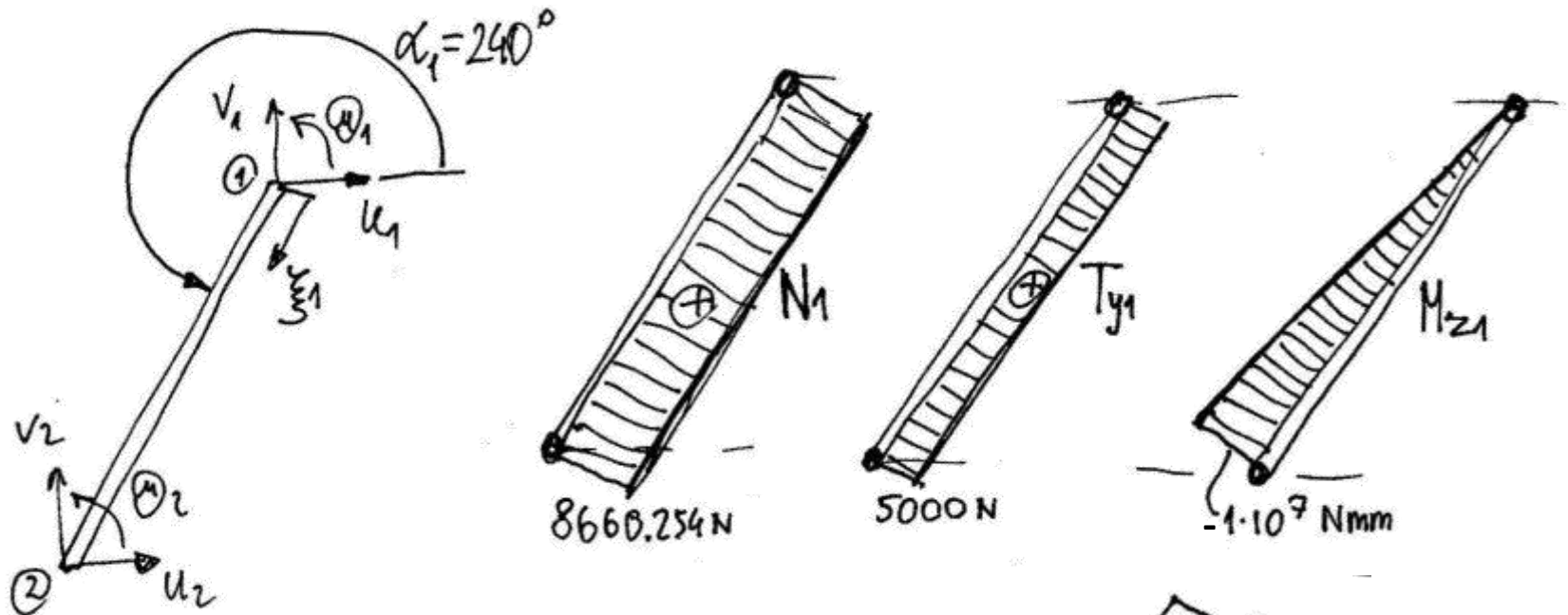
$$T_{y_1} = -EJ_2 \cdot W_1''' = -EJ_2 \cdot \underset{1 \times 4}{[N''']} \cdot \begin{Bmatrix} q_2 \\ q_3 \\ q_5 \\ q_6 \end{Bmatrix}_1 =$$

$$= -EJ_2 \left[\frac{12}{l_1^3}, \frac{6}{l_1^2}, -\frac{12}{l_1^3}, \frac{6}{l_1^2} \right] \cdot \begin{Bmatrix} q_2 \\ q_3 \\ q_5 \\ q_6 \end{Bmatrix}_1 = 5000 \text{ N}$$



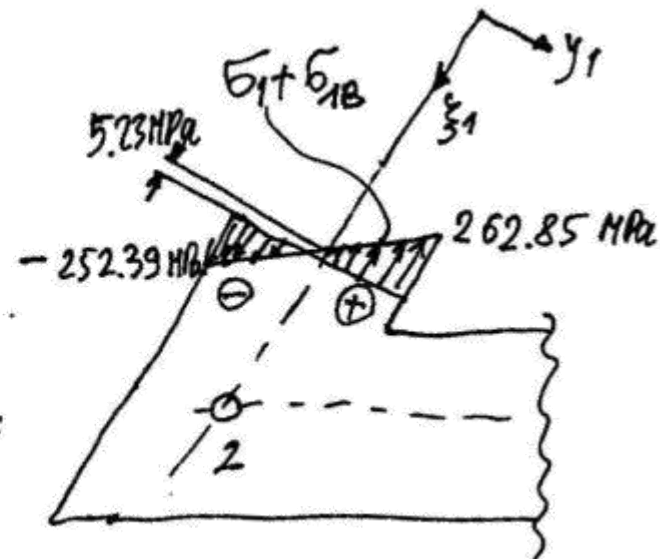
Internal forces and stresses in element 1 (resultant)

1



$$\sigma_{1B}\left(\frac{x_1}{l_1}, y_1\right) = - \frac{M_{z1}\left(\frac{x_1}{l_1}\right) \cdot y_1}{J_z}$$

$$\sigma_{1B}\left(2l_1, \frac{h}{2}\right) = \frac{-(-1 \cdot 10^7 \text{ Nmm}) \cdot \frac{h}{2}}{J_z} = 257.62 \text{ MPa}$$



Internal stresses and forces in element 2:

$$\boxed{2} \quad \begin{Bmatrix} q_1 \\ q_2 \\ q_3 \\ q_4 \\ q_5 \\ q_6 \end{Bmatrix}_2 = [T_f]_2 \cdot \begin{Bmatrix} q_9 \end{Bmatrix}_2 = [T_f]_2 \cdot \begin{Bmatrix} u_2 \\ v_2 \\ \phi_2 \\ u_3 \\ v_3 \\ \phi_3 \end{Bmatrix}_2 =$$

$$= \begin{bmatrix} 1 & 0 & 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \end{bmatrix} \cdot \begin{Bmatrix} 0 \\ -0.0604 \\ -0.01907 \\ 0 \\ -28.6692 \\ -0.03334 \end{Bmatrix}_2 = \begin{Bmatrix} 0 \\ -0.0604 \\ -0.01907 \\ 0 \\ -28.6692 \\ -0.03334 \end{Bmatrix}_2$$

Internal stresses and forces in element 2:

[2]

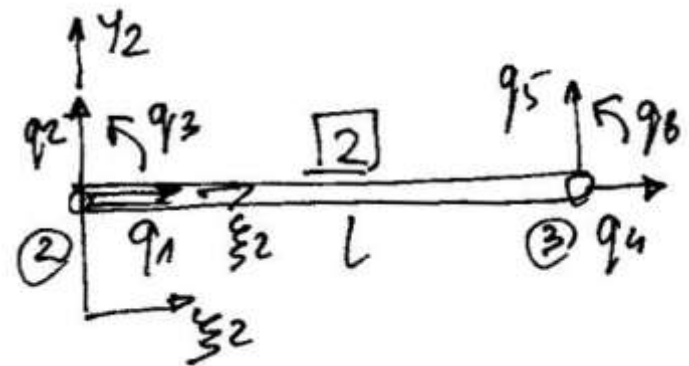
$$\begin{Bmatrix} q_1 \\ q_2 \\ q_3 \\ q_4 \\ q_5 \\ q_6 \end{Bmatrix}_2 = \begin{Bmatrix} 0 \\ -0.0604 \\ -0.01907 \\ 0 \\ -28.6692 \\ -0.03334 \end{Bmatrix}_2$$

Tension in element 2:

DAK.

$$\sigma_2 = \frac{E}{L} (q_4 - q_1)_2 = 0$$

$$N_2 = \sigma_2 \cdot A = 0$$



Internal forces in element 2 due to bending:

$$M_{z_2}(\xi_2) = EJ_2 \cdot w_2'' = EJ_2 \cdot [N]_{1 \times 4}^4 \cdot \begin{Bmatrix} q_2 \\ q_3 \\ q_5 \\ q_6 \end{Bmatrix}_2 =$$

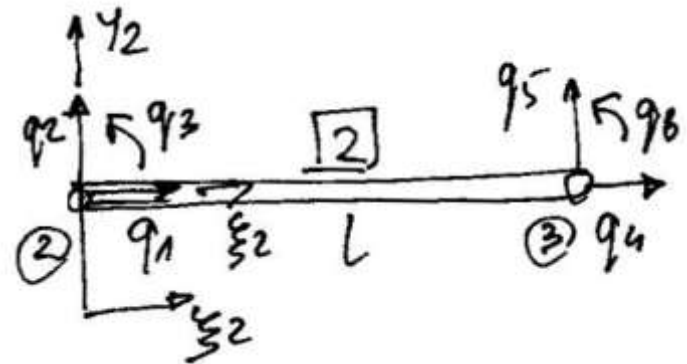
$$= EJ_2 \left[-\frac{6}{l^2} + \frac{12}{l^3} \xi_2, -\frac{4}{l} + \frac{6}{l^2} \xi_2, \frac{6}{l^2} - \frac{12}{l^3} \xi_2, -\frac{2}{l} + \frac{6}{l^2} \xi_2 \right] \cdot \begin{Bmatrix} q_2 \\ q_3 \\ q_5 \\ q_6 \end{Bmatrix}_2 \Rightarrow$$

$$M_{z_2}(0) = -1 \cdot 10^7 \text{ Nmm} ; \quad M_{z_2}(l) = 0$$

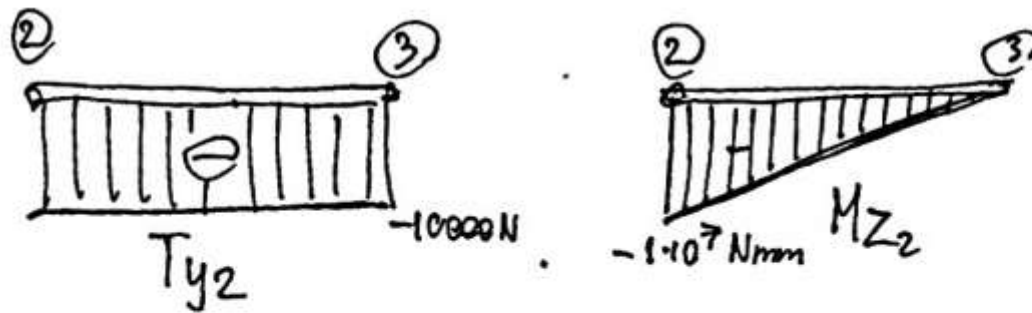
$$T_{y_2} = -EJ_2 \cdot w_2''' = -EJ_2 [N]_{1 \times 4}^4 \cdot \begin{Bmatrix} q_2 \\ q_3 \\ q_5 \\ q_6 \end{Bmatrix}_2 =$$

$$= -EJ_2 \left[\frac{12}{l^3}, \frac{6}{l^2}, -\frac{12}{l^3}, \frac{6}{l^2} \right] \cdot \begin{Bmatrix} q_2 \\ q_3 \\ q_5 \\ q_6 \end{Bmatrix}_2 =$$

$$= -10000 \text{ N}$$

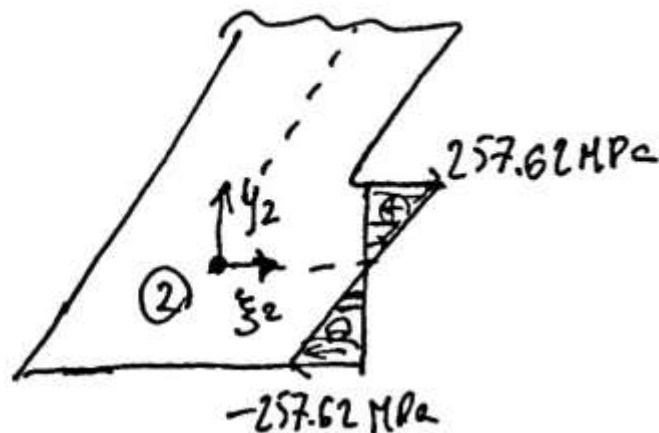


Internal forces and stresses in element 2 (resultant)



$$\sigma_{2B}(\xi_2, y_2) = -\frac{M_{z2}(\xi_2) \cdot y_2}{J_z}$$

$$\sigma_{2B}\left(0, \frac{h}{2}\right) = -\frac{(-1.10^7 \text{ Nmm}) \cdot \frac{h}{2}}{J_z} = 257.62 \text{ MPa}$$



Internal stresses and forces in element 3:

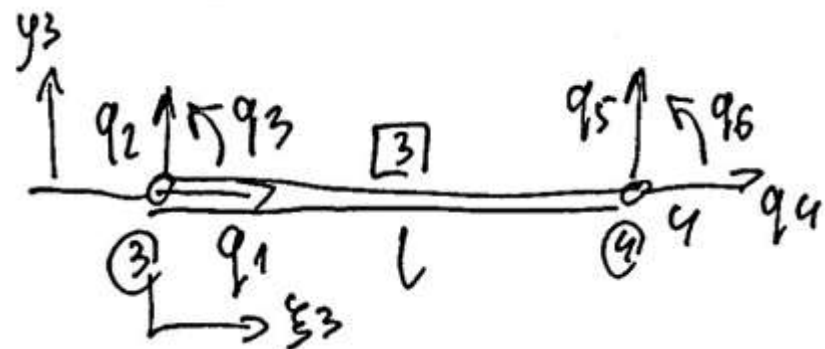
[3]

$$\begin{Bmatrix} q_1 \\ q_2 \\ q_3 \\ q_4 \\ q_5 \\ q_6 \end{Bmatrix}_3 = [T_f]_3 \cdot \begin{Bmatrix} q_9 \end{Bmatrix}_3 = [1]_3 \cdot \begin{Bmatrix} u_3 \\ v_3 \\ \theta_3 \\ u_4 \\ v_4 \\ \theta_4 \end{Bmatrix}_3 = \begin{Bmatrix} 0 \\ -28.6692 \\ -0.0334 \\ 0 \\ -62.0486 \\ -0.0334 \end{Bmatrix}_3$$

Tension in element 3:

$$\sigma_3 = \frac{E}{L} (q_4 - q_1)_3 = 0$$

$$N_3 = \sigma_3 \cdot A = 0$$

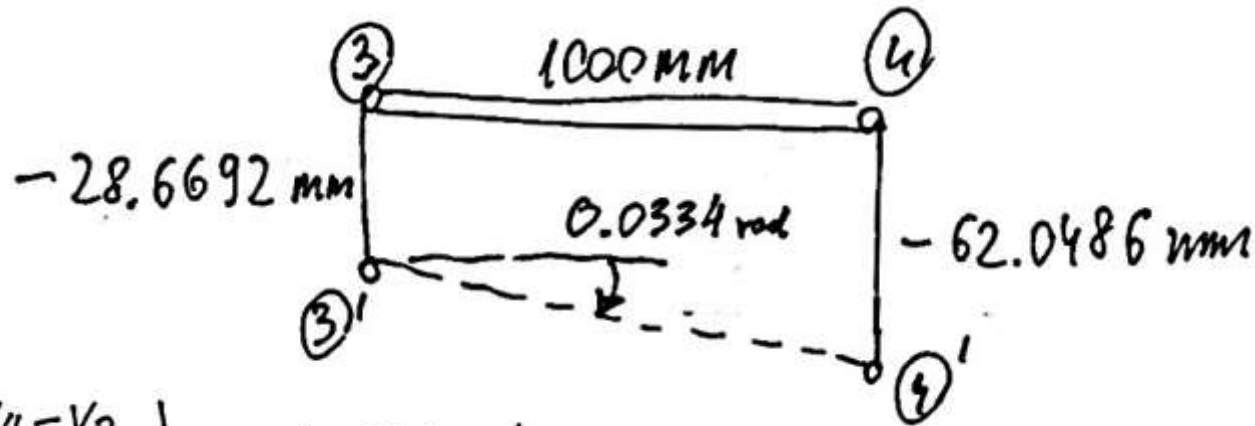


Internal forces in element 3 due to bending:

$$M_{23}(\xi_3) = EI_2 \cdot w_3'' = EI_2 \cdot \left[N'' \right]_{1 \times 4} \cdot \begin{Bmatrix} q_2 \\ q_3 \\ q_5 \\ q_6 \end{Bmatrix}_3 =$$

$$= EI_2 \left[-\frac{6}{l^2} + \frac{12}{l^3} \xi_3, -\frac{4}{l} + \frac{6}{l^2} \xi_3, \frac{6}{l^2} - \frac{12}{l^3} \xi_3, -\frac{2}{l} + \frac{6}{l^2} \xi_3 \right] \cdot \begin{Bmatrix} q_2 \\ q_3 \\ q_5 \\ q_6 \end{Bmatrix}_3 =$$

$$= 0 \Rightarrow \sigma_{3B}(\xi_3, y_3) = 0$$



$$\arctan\left(\frac{v_4 - v_3}{l}\right) = -0.0334 \text{ rad}$$

Equilibrium:

